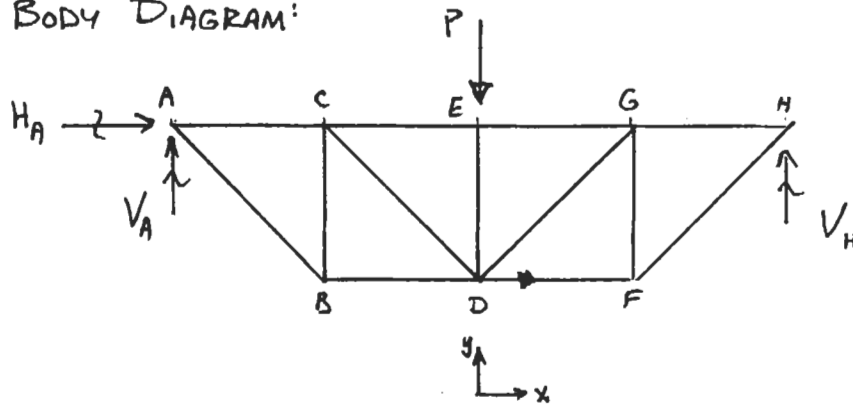


MX 8

FREE BODY DIAGRAM:



APPLY EQUILIBRIUM TO FIND REACTIONS

$$\rightarrow \sum F_x = 0$$

$$\boxed{H_A = 0 \text{ N}}$$

$$+\uparrow \sum F_y = 0$$

$$V_A + V_H - P = 0$$

$$+(\sum M_A = 0$$

$$V_H(4L) - P(2L) = 0$$

$$\boxed{V_H = \frac{P}{2}}$$

$$\Rightarrow \boxed{V_A = \frac{P}{2}}$$

TO DETERMINE THE DEFLECTION OF D, WE NEED TO EMPLOY COMPATIBILITY + CONSTITUTIVE LAWS.

OUR CONSTITUTIVE LAW FOR BAR DEFORMATION IS:

$$\delta_{ij} = \frac{F_{ij} L_{ij}}{AE}$$

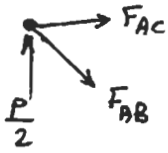
SO WE'LL NEED TO SOLVE FOR THE BAR FORCES IN ORDER TO DETERMINE THEIR EXTENSIONS, AND HENCE THE TRUSS DEFLECTION.

BECAUSE OF SYMMETRY, I ONLY NEED TO FIND HALF OF THE BAR FORCES. ALL OF THE PAIRS MIRRORED IN THE D-E AXIS WILL HAVE THE SAME BAR FORCE:

$$\begin{aligned} F_{AC} &= F_{CH} & F_{BD} &= F_{DC} \\ F_{AB} &= F_{BH} & F_{CD} &= F_{DG} \\ F_{BC} &= F_{CG} & F_{CE} &= F_{EG} \end{aligned}$$

SOLVE FOR INDEPENDENT BAR FORCES:

MOJ @ A:



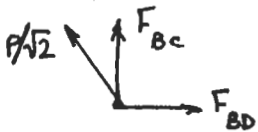
$$\sum F_y = \frac{P}{2} - F_{AB} \cos 45 = 0$$

$$F_{AB} = \frac{P}{\sqrt{2}}$$

$$\sum F_x = F_{AC} + F_{AB} \sin 45 = 0$$

$$F_{AC} = -\frac{P}{2}$$

MOJ @ B:



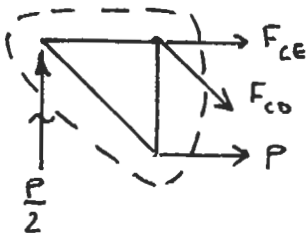
$$\sum F_y = 0 = F_{BC} + (P/\sqrt{2}) \cos 45$$

$$F_{BC} = -P/2$$

$$\sum F_x = 0 = F_{BD} - (P/\sqrt{2}) \sin 45$$

$$F_{BD} = P/2$$

MOS:



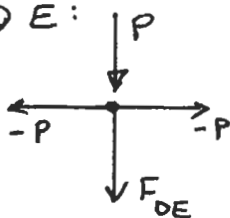
$$\sum F_y = 0 = \frac{P}{2} - F_{CD} \cos 45$$

$$F_{CD} = \frac{P}{\sqrt{2}}$$

$$\sum M_D = 0 = -F_{CE} \cdot k - \frac{P}{2} \cdot (\frac{1}{2}) = 0$$

$$F_{CE} = -P$$

MOJ @ E:



$$\sum F_y = 0$$

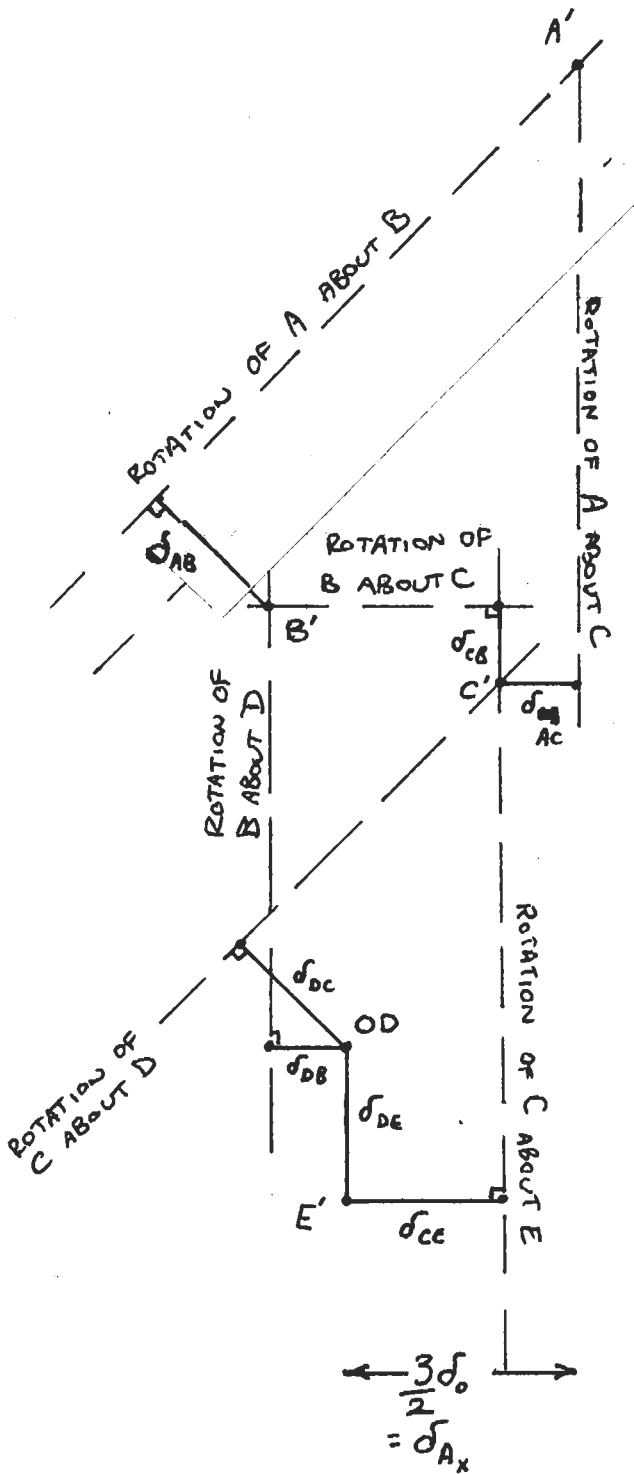
$$-P - F_{DE} = 0$$

$$F_{DE} = -P$$

BAR	FORCE $\left(\frac{F_{ij}}{P}\right)$	LENGTH $\left(\frac{L_{ij}}{L}\right)$	DEFORMATION $\frac{\delta_{ij}}{PL/AE}$
AB	$+\frac{1}{\sqrt{2}}$	$\sqrt{2}$	$+1$
AC	$-\frac{1}{2}$	1	$-\frac{1}{2}$
CB	$-\frac{1}{2}$	1	$-\frac{1}{2}$
CE	-1	1	-1
CD	$+\frac{1}{\sqrt{2}}$	$\sqrt{2}$	$+1$
BD	$+\frac{1}{2}$	1	$+\frac{1}{2}$
ED	-1	1	-1
EG	-1	1	-1
DG	$+\frac{1}{\sqrt{2}}$	$\sqrt{2}$	$+1$
DF	$+\frac{1}{2}$	1	$+\frac{1}{2}$
GF	$-\frac{1}{2}$	1	$-\frac{1}{2}$
GH	$-\frac{1}{2}$	1	$-\frac{1}{2}$
FH	$+\frac{1}{\sqrt{2}}$	$\sqrt{2}$	$+1$

NOW WE CAN GO AHEAD AND PLOT OUR TRUSS DEFLECTION DIAGRAM.

$$\delta_o = \frac{PL}{AE}$$



IF MY HINGE POINT A' ENDS UP DISPLACED FROM MY ORIGIN BY δ_{Ax} AND δ_{Ay} , THEN MY ORIGIN OD IS DISPLACED FROM A' BY $-\delta_{Ax}$ AND $-\delta_{Ay}$.

$$\frac{13\delta_o}{2} = \delta_{Ay}$$

IF I NOW CONSIDER THE FIXED FRAME OA, WHERE A AND A' ARE THE SAME, I CAN FIND THE DEFLECTION OF D' IN THE FIXED FRAME, WHICH IS JUST ITS DISPLACEMENT FROM A', NAMELY $-\delta_{Ax} \hat{i} - \delta_{Ay} \hat{j}$.

THE JOINT D WILL TRANSLATE DOWN BY $\frac{13 PL}{2 AE}$, AND LEFT BY $\frac{3 PL}{2 AE}$

ESTIMATE OF TRUSS DEFLECTIONS

BARS IN EXPERIMENTAL TRUSS MADE OF STEEL

- HOLLOW WITH 22 mm OUTER DIAMETER AND

1.5 mm WALL THICKNESS (IGNORE END FITTINGS)

$$A \approx 2\pi r t \approx 10.5 \times 10^{-3} \times 2 \times \pi \times 1.5 \times 10^{-3} \\ \approx 100 \text{ mm}^2$$

$$L = 0.5 \text{ m}$$

$$E = 210 \text{ GPa}$$

CENTER POINT DEFLECTION

$$\frac{\delta_D}{P} = \left(\frac{0.5}{100 \times 10^{-6} \times 210 \times 10^9} \right) \left(\frac{-13\hat{j} - 3\hat{i}}{2} \right)$$

$$\boxed{\frac{\delta_D}{P} = -\frac{3.1 \times 10^{-7}}{2} \hat{j} - \frac{7.1 \times 10^{-8}}{2} \hat{i} \quad \text{N/m}}$$